

# AUV

## Modeling, Control, Obstacle avoidance

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# Outlines

- ❑ Coordinate system, general 6Dof inertial and relative
- ❑ Modeling MIMO system using Fourier Integral method
- ❑ MIMO Control design
  - Specifications
  - Loop shaping
- ❑ Optimal path planner using the DIDO solver
- ❑ Experiments

# Coordinate System

## Technical terms (terminology)

### x

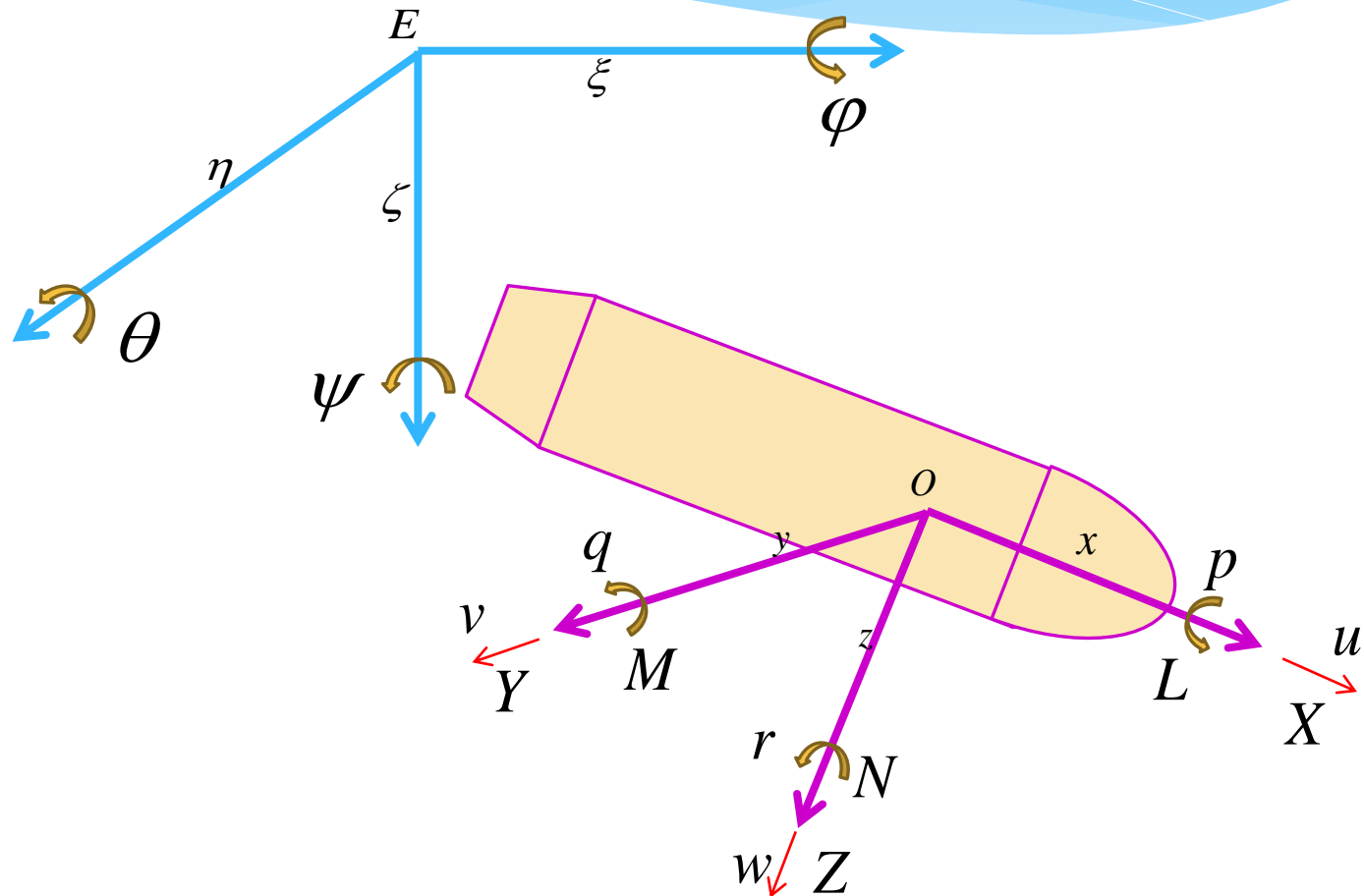
- \* Surge:  $u$ ,  $X$
- \* Roll:  $p$ ,  $L$

### y

- \* Sway:  $v$ ,  $Y$
- \* Pitch:  $q$ ,  $M$

### z

- \* Heave:  $w$ ,  $Z$
- \* Yaw:  $r$ ,  $N$



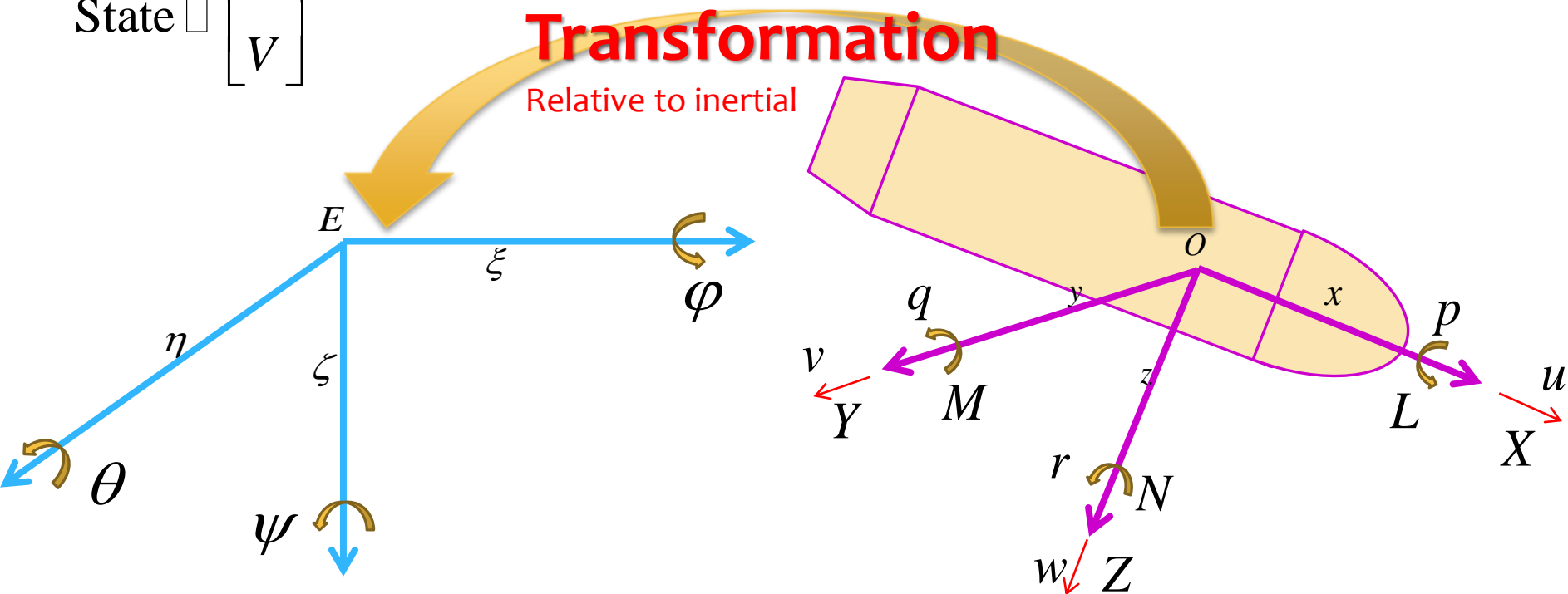
# State Control

## AUV State

$$R = [r' \parallel \Lambda']' = [\xi \quad \eta \quad \zeta \parallel \varphi \quad \theta \quad \psi]'$$

$$V = [U' \parallel \Omega']' = [u \quad v \quad w \parallel p \quad q \quad r]'$$

$$\text{State} \square \begin{bmatrix} R \\ V \end{bmatrix}$$



# Transformation

- Transformation from fixed-body frame to inertial coordinate system

$$\text{Inertial velocity} \begin{bmatrix} \dot{\xi}_G \\ \dot{\eta}_G \\ \dot{\zeta}_G \\ \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix} = \begin{bmatrix} T_1 & 0_3 \\ 0_3 & T_2 \end{bmatrix} \begin{bmatrix} u \\ v \\ w \\ p \\ q \\ r \end{bmatrix} \text{Relative velocity}$$

- Transform matrixes

$$T_1 = \begin{bmatrix} \cos \psi \cos \theta & \cos \psi \sin \theta \sin \varphi - \sin \psi \cos \varphi & \cos \psi \sin \theta \cos \varphi + \sin \psi \sin \varphi \\ \sin \psi \cos \theta & \sin \psi \sin \theta \sin \varphi + \cos \psi \cos \varphi & \sin \psi \sin \theta \cos \varphi - \cos \psi \sin \varphi \\ -\sin \theta & \cos \theta \sin \varphi & \cos \theta \cos \varphi \end{bmatrix}$$

$$T_2 = \begin{bmatrix} 1 & \tan \theta \sin \varphi & \tan \theta \cos \varphi \\ 0 & \cos \varphi & -\sin \varphi \\ 0 & \sin \varphi \sec \theta & \cos \varphi \sec \theta \end{bmatrix}$$

# Momentum Theorem

*momentum*  $\triangleq$  *Mass* · *Velocity* (scalar product)

*external forces and moments*  $= \frac{d}{dt}$  [*momentum*]

Origin is located at the center of buoyancy and the center of gravity lies at the point  $r_G$ .

➤ *Newton's equation of motion*  $\sum F = ma_G$

$a_G \triangleq$  center mass acceleration,  $m \triangleq$  body mass,  $\sum F \triangleq$  external forces

$$a_G = \frac{\partial U}{\partial t} + \Omega \times U + \dot{\Omega} \times r_G + \Omega \times \Omega \times r_G$$

$$\Omega = [p, q, r]' \quad , \quad U = [u, v, w]'$$

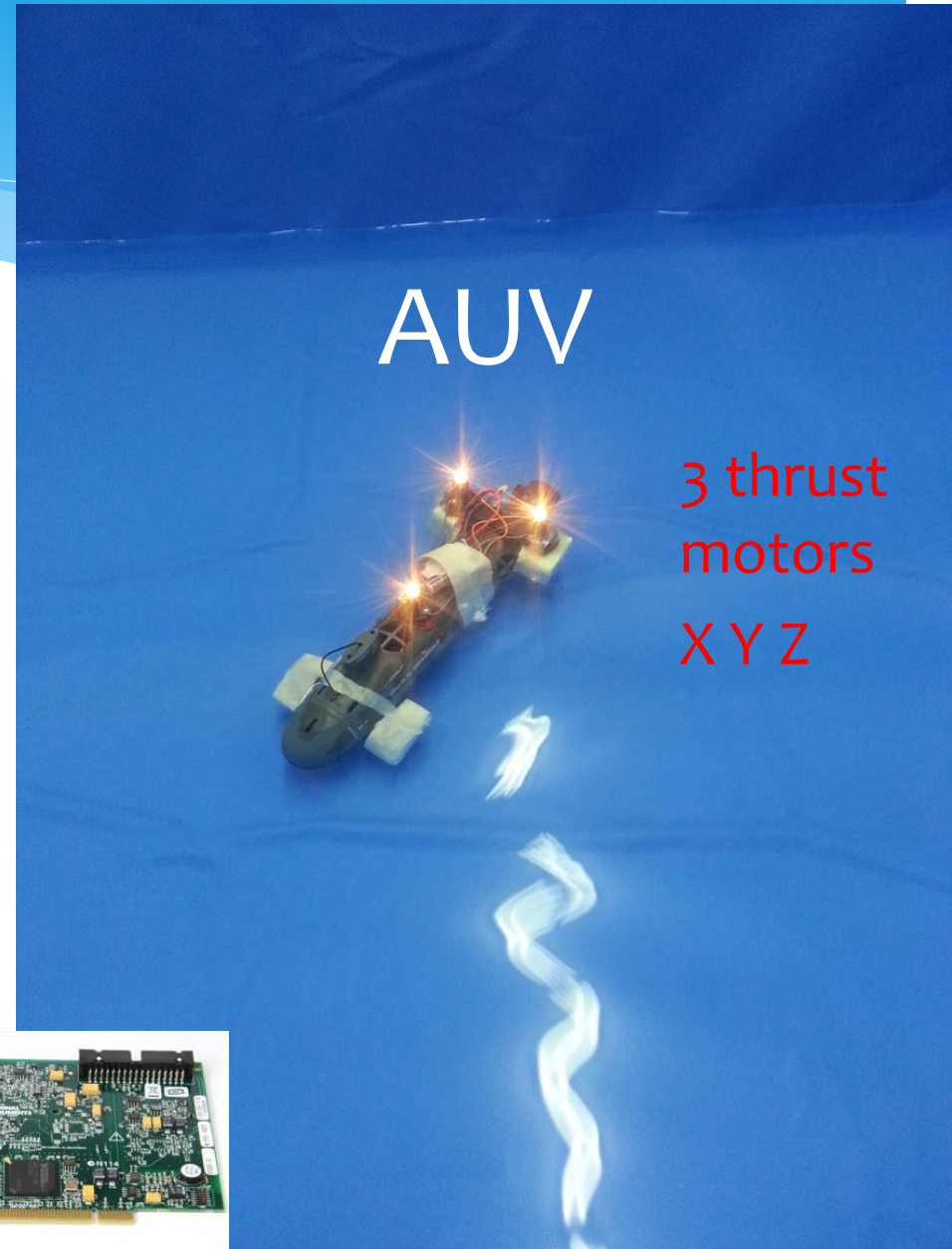
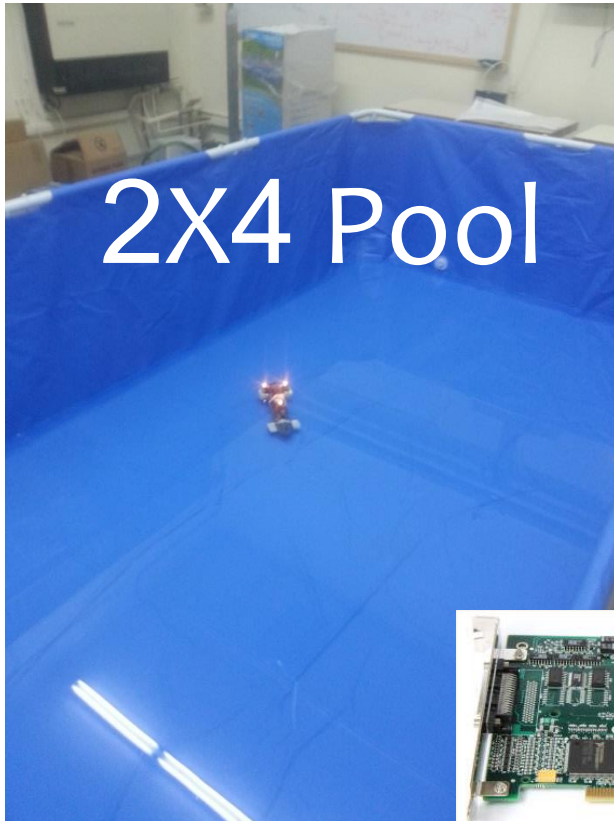
➤ *Euler's equation of motion*  $\sum M_B = \dot{H}_G + r_G \times ma_G$

$\dot{H}_G \triangleq$  rate of change of angular momentum about the center of gravity,  $\sum M_B \triangleq$  external moments

$$\dot{H}_G = [I]\dot{\Omega} + \Omega \times [I]\Omega$$

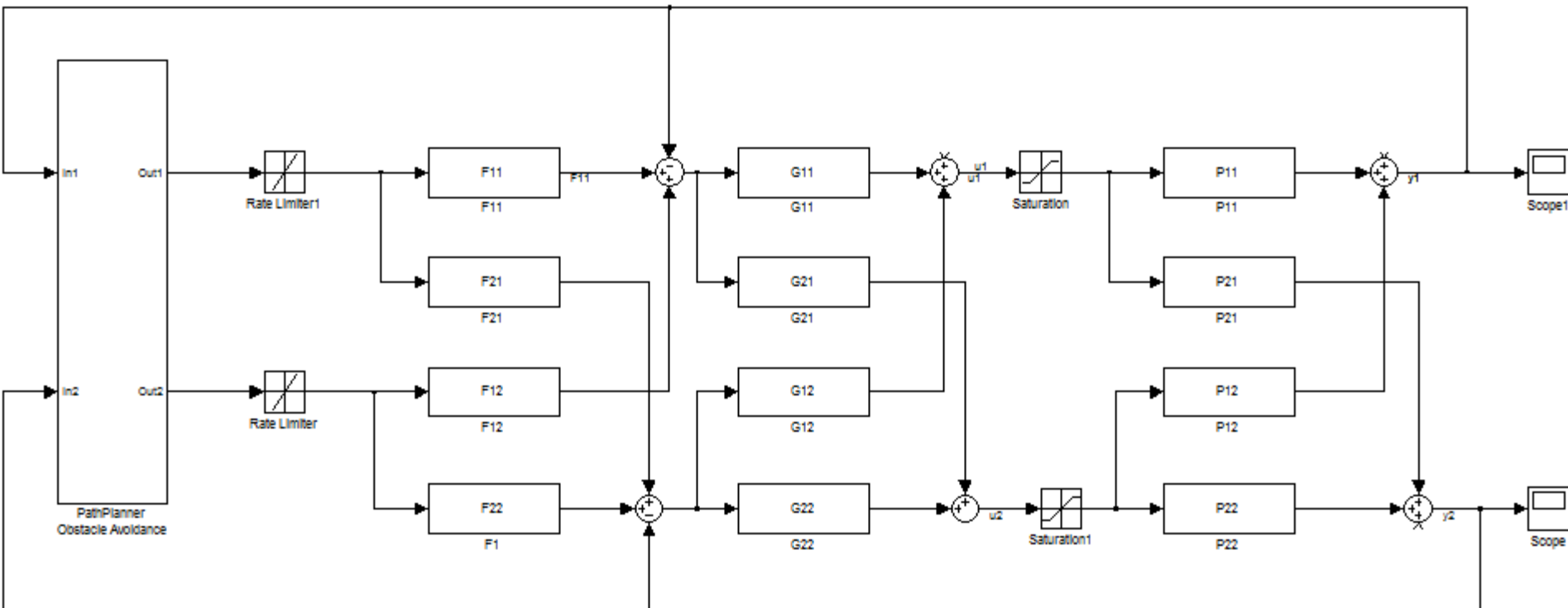
$$[I] \triangleq \text{diagonal inertia matrix} \quad ( \text{diag}[I_x \ I_y \ I_z] )$$

# Laboratory Setup



# Control Architecture

- 2 Dof MIMO:
- Path planner (MPC)
- F is the prefilter, G is the controller, P is the plant.
- Additions:
  - rate limit on the position reference command
  - Saturation on the control signal





# Modeling

□ Modeling using Fourier integral method

□ Execute on all variations of u and y:

- From u1 to y1 and y2
- From u2 to y2 and y1

$$\square \text{Plant}(j\omega) = \delta$$

$$|\text{Plant}(j\omega)| = \frac{B}{A}$$

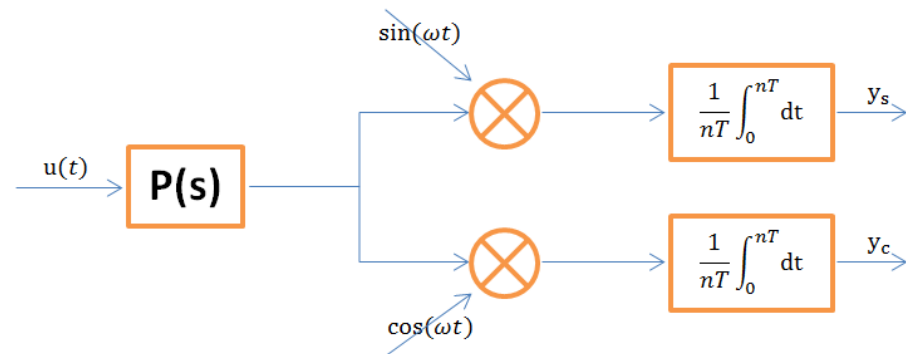
$$\begin{aligned} y_s &= \frac{1}{nT} \int_0^{nT} y(t) \sin(\omega t) dt = \frac{1}{nT} \int_0^{nT} (B \sin(\omega t + \delta) + e(t)) \sin(\omega t) dt = \\ &= \frac{1}{nT} \int_0^{nT} B \sin(\omega t + \delta) \sin(\omega t) dt + \frac{1}{nT} \int_0^{nT} e(t) \sin(\omega t) dt = \\ &= \frac{B}{2} \cos(\delta) - \underbrace{\frac{B}{2nT} \int_0^{nT} \cos(\omega t + \delta) dt}_{=0} + \underbrace{\frac{1}{nT} \int_0^{nT} e(t) \sin(\omega t) dt}_{=0, \text{uncorrelated signals}} \end{aligned}$$

$$y_s = \frac{B}{2} \cos(\delta)$$

$$y_c = \frac{B}{2} \sin(\delta)$$

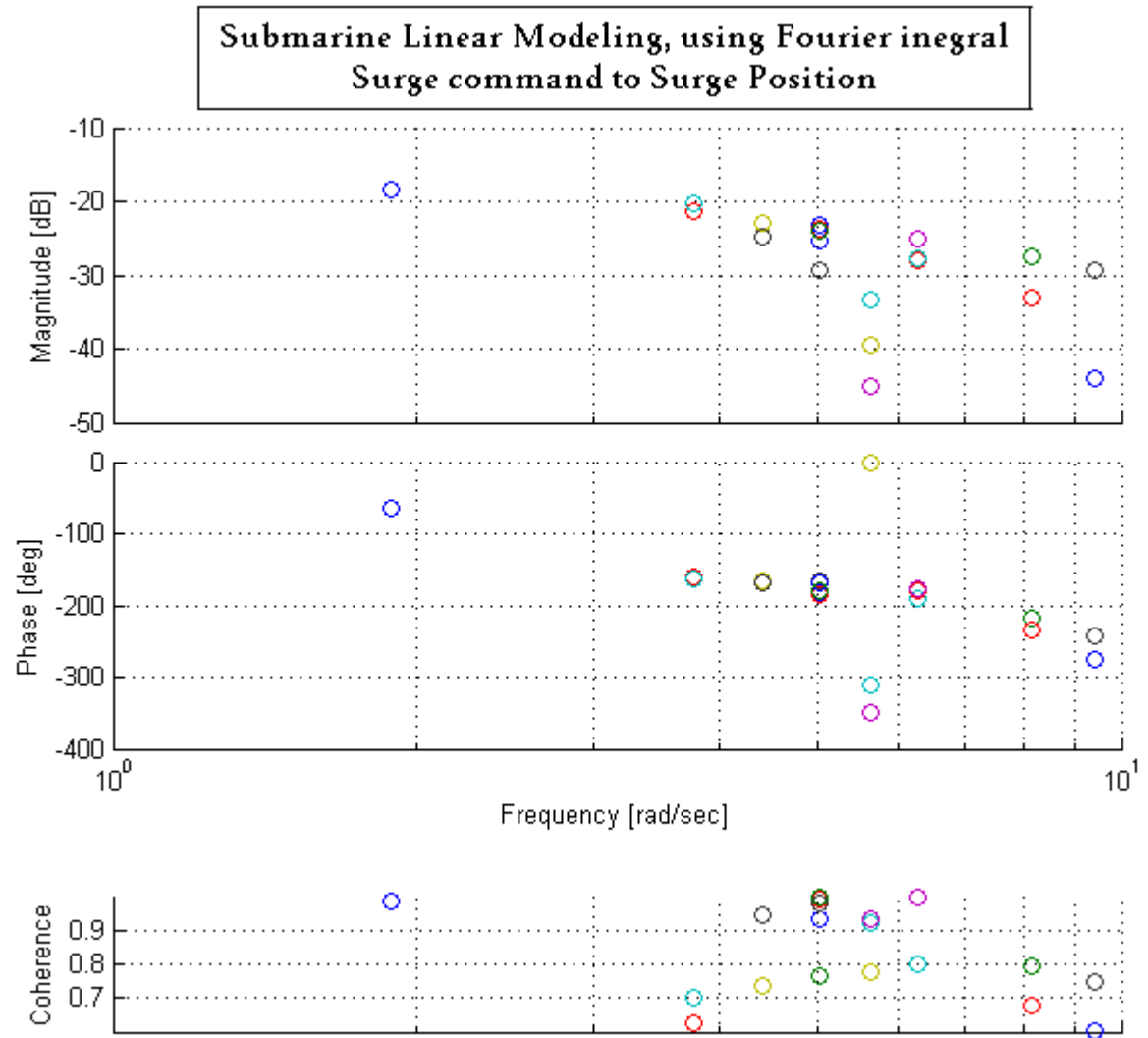
$$\delta = \arctan 2(y_s, y_c)$$

$$B = 2\sqrt{y_s^2 + y_c^2}$$



# Modeling

- Modeling results (1/4 of the plants)
- From Thrust motor input to Longitude position



# Control Design

## □ Specifications:

- stabilize the close loop
- Step response
  - Settling time 10 [s]
  - Overshoot < 10%
- Disturbance rejection
  - 6 [dB] for all frequencies
- Cross coupling sensitivity
  - 3 [dB] for all frequencies

All time domain specifications translated into frequency domain using 2<sup>nd</sup> order transfer function formulations.

# Control Design specifications

- Time domain specifications translated into frequency domain specifications using 2<sup>nd</sup> order formulations

$$G(s) = \frac{\omega_n}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$T_{\text{settling}} = \frac{4}{\zeta\omega_n}, \text{ for 2\% error}$$

$$OS = 1 + e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$$

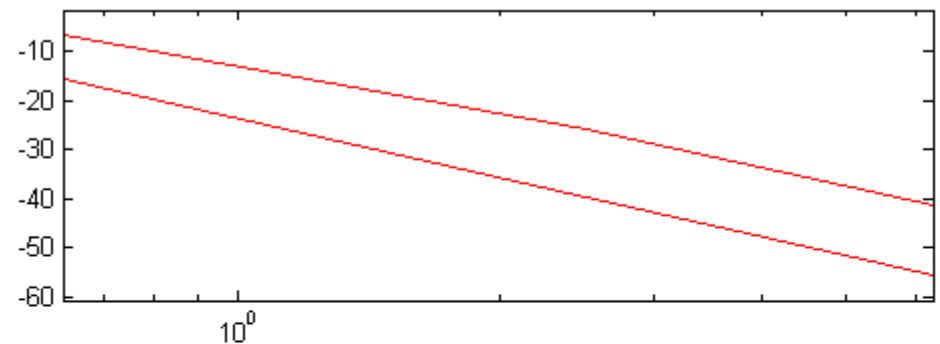
- Servo specification

$$a(\omega) \leq |ClosedLoop| \leq b(\omega)$$

- Sensitivity specification

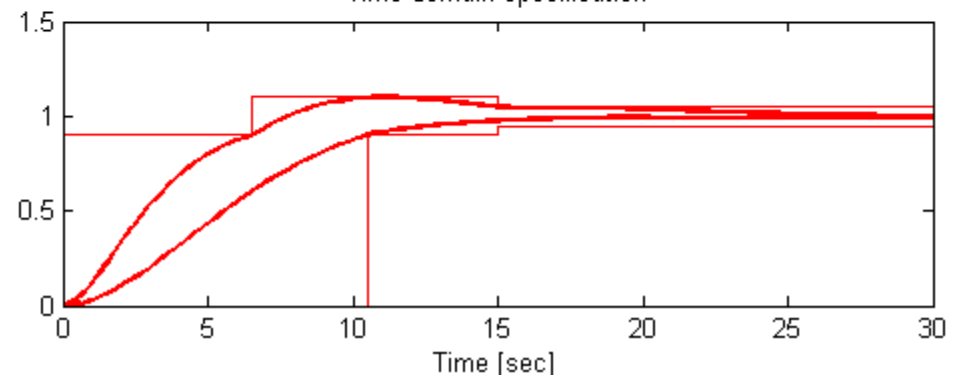
$$|sensitivity| \leq c(\omega)$$

Frequency domain specification, using 2<sup>nd</sup> order formulations



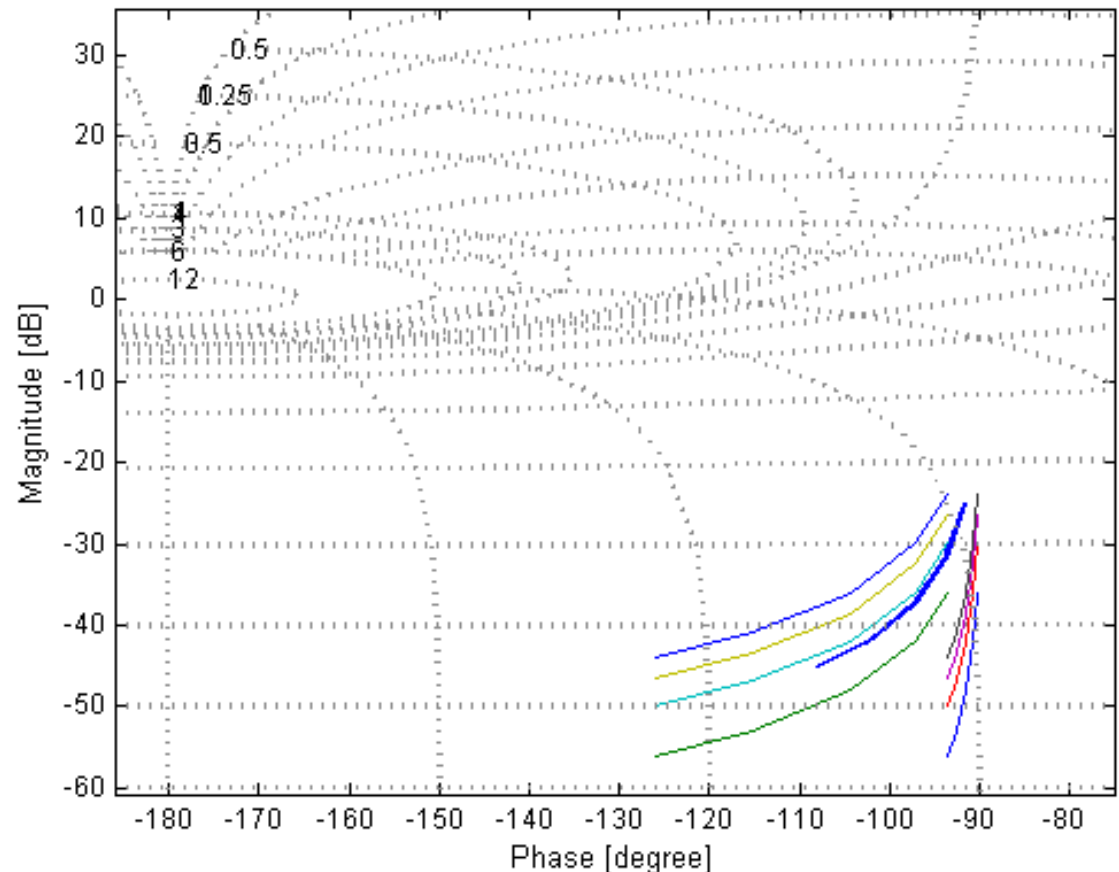
Frequency [rad/sec]

Time domain specification



# Control Design Cases

- ❑ Specifications must be satisfied for all plants (for all cases)
- ❑ The identified points from the modeling on Nichols chart
- ❑ All cases can be bounded



# Control Design Template

- The chosen linear plant structure:

$$\frac{k_{11}}{s} e^{-\tau_{11}s}$$

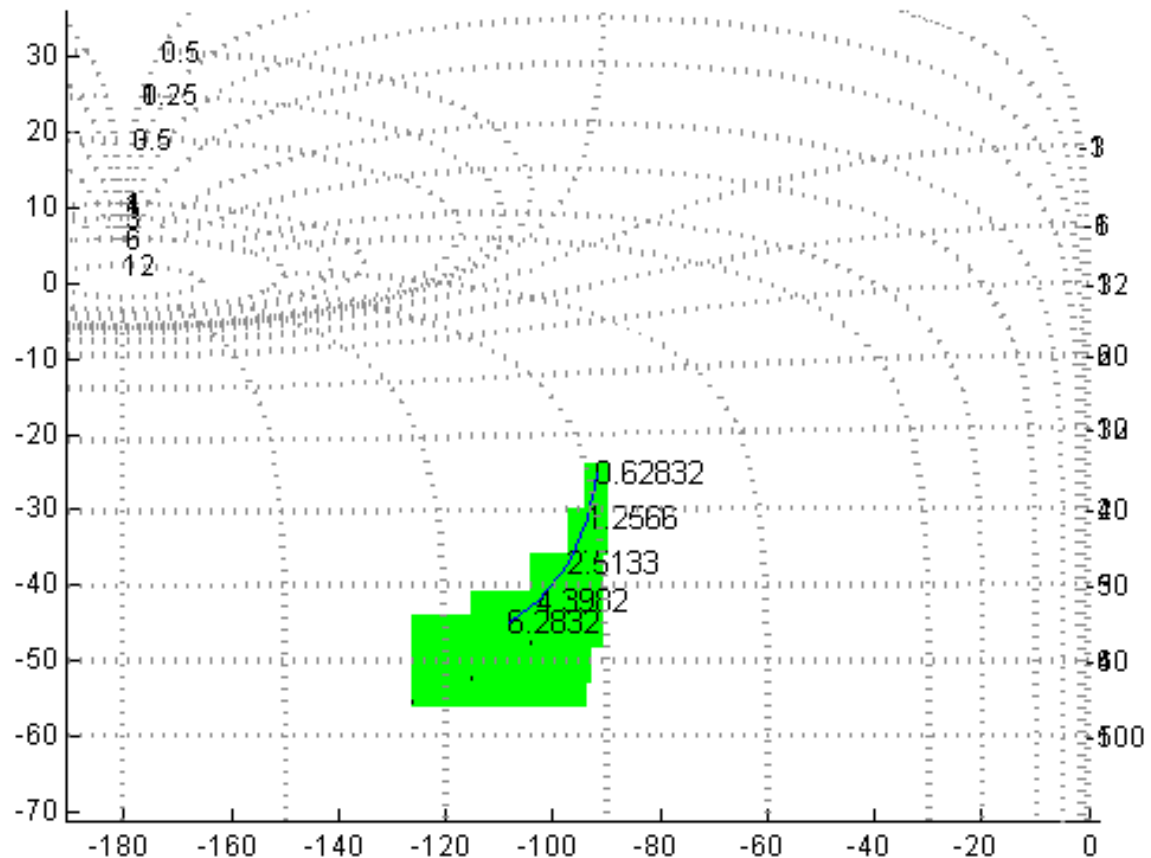
- The nominal plant is

$$\frac{0.035}{s} e^{-0.05s}$$

- The uncertainties are:

$$k_{11} \in [0.01, 0.04]$$

$$\tau_{11} \in [0.01, 0.1]$$



# Control Design

## Horowitz-Sidi bounds

### □ Tolerance bounds

$$\left| \frac{\bar{S}_{\max}}{\bar{S}_{\min}} \right| = \frac{\max_i \left| \frac{P_i G}{1 + P_i G} \right|}{\min_i \left| \frac{P_i G}{1 + P_i G} \right|} \leq \frac{b}{a}$$

### □ Sensitivity bounds

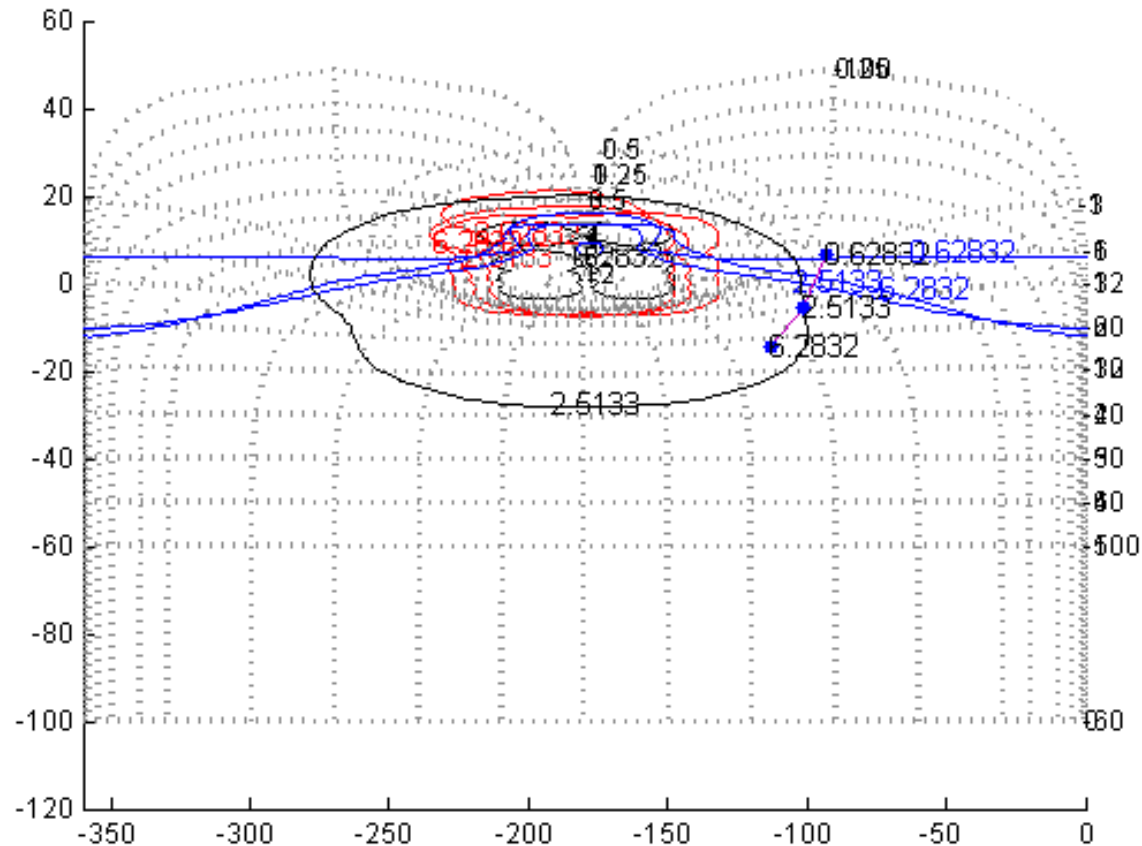
$$|S| = \left| \frac{1}{1 + PG} \right| \leq c$$

### □ Cross-Coupling bounds (only for MIMO)

$$\left| \frac{\frac{W_{12} b_{22}}{W_{11}}}{1 + \frac{L_{10}}{W_{11}}} \right| \leq b_{12}$$

# Control Design

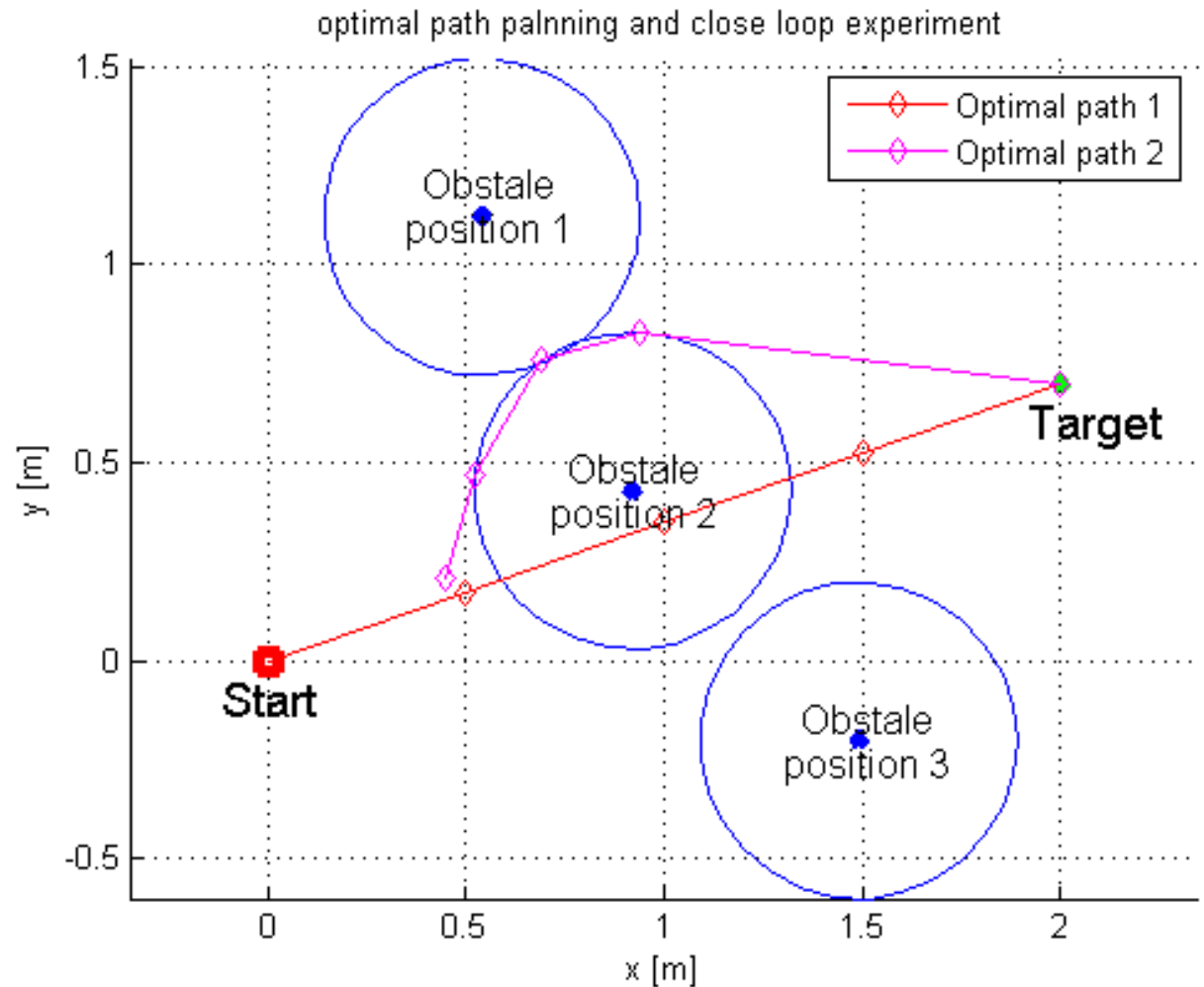
- Loop Shaping with the Horowitz-Sidi bounds
- Performance
- Sensitivity
- Cross coupling





# Experiments

- ❑ Optimal Path Planner, using DIDO solver
- ❑ The obstacle is moving
- ❑ Optimal Path 2 is the updated path



# Experiments

Reference command tracking

In this example

